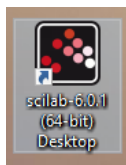
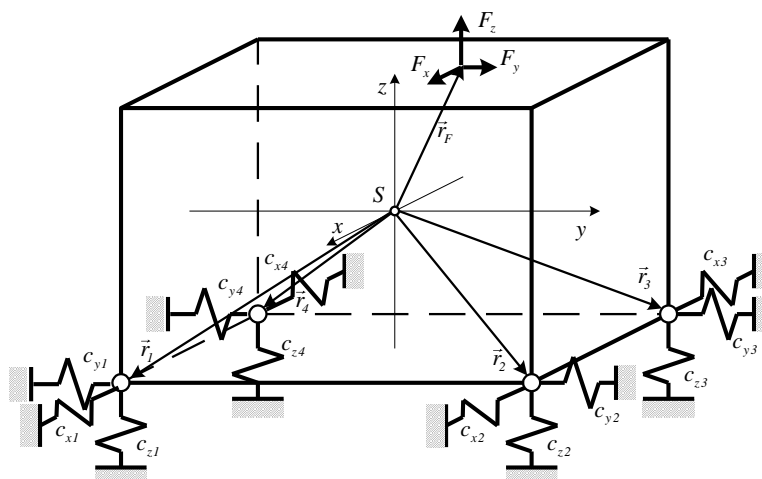


Dynamics of Machines Week 10 – Exercise



Vibration Analysis of 3D Model of a Machine and Machine Base



$$\text{Equation of motion: } \ddot{a}_i + 2\xi_i \alpha_i \dot{a}_i + \alpha_i^2 a_i = Q_{ai} \quad (i = 1, 2, 3, 4, 5, 6)$$

$$\text{Rearrangement: } \ddot{a}_i = -2\xi_i \alpha_i \dot{a}_i - \alpha_i^2 a_i + Q_{ai} \quad (i = 1, 2, 3, 4, 5, 6)$$

Mass of the machine and machine base is $m = 500 \text{ kg}$, non zero elements of the inertia tensor $J_{Sx} = 1900 \text{ kgm}^2$, $J_{Sy} = 1700 \text{ kgm}^2$, $J_{Sz} = 1500 \text{ kgm}^2$, spring constant values are the same at every vertices $c_{xi} = 5 \cdot 10^{-5} \text{ m/N}$, $c_{yi} = 5 \cdot 10^{-5} \text{ m/N}$, $c_{zi} = 2 \cdot 10^{-5} \text{ m/N}$, ($i = 1, 2, 3, 4$).

Coordinates of the vertices:

$$(x_1 = 0,6 \text{ m}, y_1 = -0,7 \text{ m}, z_1 = -0,4 \text{ m}), (x_2 = 0,6 \text{ m}, y_2 = 0,8 \text{ m}, z_2 = -0,4 \text{ m}),$$

$$(x_3 = -0,5 \text{ m}, y_3 = 0,8 \text{ m}, z_3 = -0,4 \text{ m}), (x_4 = -0,5 \text{ m}, y_4 = -0,7 \text{ m}, z_4 = -0,4 \text{ m}).$$

Excitation angular frequency: $\omega = 62,8 \text{ (rad / s)}$

Coordinates of excitation force: $F_z = 500 \cos(62,8 t)$

Coordinates of the position vector of excitation force, \vec{r}_F : $x_F = -0,3 \text{ m}$, $y_F = 0,2 \text{ m}$, $z_F = 0,4 \text{ m}$

Lehr damping coefficient in every natural frequency: $\xi = 0,1$

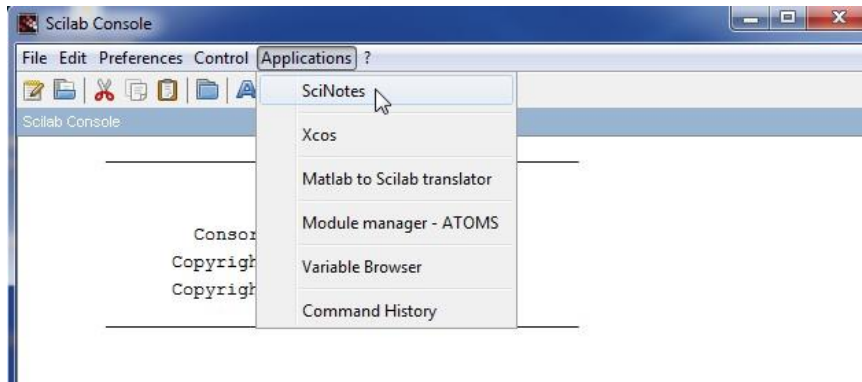
Tasks: Determine the displacement of point P as a function of time!

Position vector of point P : $\vec{r}_p = (0,5 \vec{i} + 0,6 \vec{j} + 0,4 \vec{k}) \text{ m}$

Final time $t_{max} = 15 \text{ (s)}$. The machine revs up from rest to the working engine speed during $t_{start} = 3 \text{ (s)}$.

Length of a time step is $\Delta t = 0,001 \text{ (s)}$.

Solving method: The problem will be solved in two steps. In the first step the undamped natural frequencies (eigenvalues) and eigenvectors will be determined and in the second step the uncoupled (independent) damped differential equations will be solved with Runge-Kutta method.



// week 10 - exercise

`clear;`

`usecanvas(%F);`

`//stacksize('max')`

// Variables:

// mass (kg) and moment of inertia variables (kgm²)

`m=500; Jsx=1900; Jsy=1700; Jsx=1500;`

// spring constants (m/N)

`cxi=5.0e-5; cyi=5.0e-5; czi=2.0e-5;`

// coordinates of the 4 vertices (m)

`x1= 0.6; y1=-0.7; z1=-0.4;`

`x2= 0.6; y2= 0.8; z2=-0.4;`

`x3=-0.5; y3= 0.8; z3=-0.4;`

`x4=-0.5; y4=-0.7; z4=-0.4;`

// Coordinates of the position vector of excitation force (m)

`xF=-0.3; yF= 0.2; zF= 0.4;`

// Coordinates of the position vector of point P

`xP=+0.5; yP= 0.6; zP= 0.4;`

// Lehr damping in every natural frequency

`xxi=zeros(6,1);`

`xxi(1,1)=0.1;`

`xxi(2,1)=0.1;`

`xxi(3,1)=0.1;`

`xxi(4,1)=0.1;`

`xxi(5,1)=0.1;`

`xxi(6,1)=0.1;`

`F=zeros(3,1); F(3,1)= 500;`

// coordinates of the force vector (N)

`omega=62.8;`

// excitation angular frequency (rad/s)

// Initial conditions matrix (initial displacement and initial velocity values)

`abi0(1,1)=0;abi0(2,1)=0;abi0(3,1)=0;abi0(4,1)=0;abi0(5,1)=0;abi0(6,1)=0;`

`abi0(7,1)=0;abi0(8,1)=0;abi0(9,1)=0;abi0(10,1)=0;abi0(11,1)=0;abi0(12,1)=0;`

// time variables:

```
tmax=15;           // final time (s)
tstart=3;          // (s)
dt=0.001;         // time step (s)
t0=0;             // initial time (s)
t=0:dt:tmax;      // time interval (s)
```

// multiplier matrices of position vectors

```
R1=zeros(3,3);
R1(1,2)= z1; R1(1,3)=-y1;
R1(2,1)=-z1; R1(2,3)= x1;
R1(3,1)= y1; R1(3,2)=-x1;
R2=zeros(3,3);
R2(1,2)= z2; R2(1,3)=-y2;
R2(2,1)=-z2; R2(2,3)= x2;
R2(3,1)= y2; R2(3,2)=-x2;
R3=zeros(3,3);
R3(1,2)= z3; R3(1,3)=-y3;
R3(2,1)=-z3; R3(2,3)= x3;
R3(3,1)= y3; R3(3,2)=-x3;
R4=zeros(3,3);
R4(1,2)= z4; R4(1,3)=-y4;
R4(2,1)=-z4; R4(2,3)= x4;
R4(3,1)= y4; R4(3,2)=-x4;
```

$$\underline{\underline{R}}_i = \begin{bmatrix} 0 & z_i & -y_i \\ -z_i & 0 & x_i \\ y_i & -x_i & 0 \end{bmatrix}$$

```
RF=zeros(3,3);
RF(1,2)= zF; RF(1,3)=-yF;
RF(2,1)=-zF; RF(2,3)= xF;
RF(3,1)= yF; RF(3,2)=-xF;
```

$$\underline{\underline{M}} = \begin{bmatrix} \underline{\underline{m}} & \underline{\underline{0}} \\ \underline{\underline{0}} & \underline{\underline{J}}_{\underline{\underline{S}}} \end{bmatrix} = \begin{bmatrix} m & 0 & 0 & 0 & 0 & 0 \\ 0 & m & 0 & 0 & 0 & 0 \\ 0 & 0 & m & 0 & 0 & 0 \\ 0 & 0 & 0 & J_{Sx} & 0 & 0 \\ 0 & 0 & 0 & 0 & J_{Sy} & 0 \\ 0 & 0 & 0 & 0 & 0 & J_{Sz} \end{bmatrix}$$

```
RP=zeros(3,3);
RP(1,2)= zP; RP(1,3)=-yP;
RP(2,1)=-zP; RP(2,3)= xP;
RP(3,1)= yP; RP(3,2)=-xP;
```

// mass matrix

```
M=zeros(6,6);
M(1,1)=m; M(2,2)=m; M(3,3)=m;
M(4,4)=Jsx; M(5,5)=Jsy; M(6,6)=Jsz;
MM=M;
```

$$\underline{\underline{C}}_i = \begin{bmatrix} \frac{I}{c_{xi}} & 0 & 0 \\ 0 & \frac{I}{c_{yi}} & 0 \\ 0 & 0 & \frac{I}{c_{zi}} \end{bmatrix}$$

// spring matrix

```
C=zeros(6,6);
Ci=zeros(3,3);
Ci(1,1)=1/cxi; Ci(2,2)=1/cyi; Ci(3,3)=1/czi;
```

///// stiffness matrix

```
C(1:3,1:3)=Ci+Ci+Ci+Ci;
C(1:3,4:6)=Ci*R1+Ci*R2+Ci*R3+Ci*R4;
C(4:6,1:3)=R1'*Ci+R2'*Ci+R3'*Ci+R4'*Ci;
C(4:6,4:6)=R1'*Ci*R1+R2'*Ci*R2+R3'*Ci*R3+R4'*Ci*R4;
CC=C;
```

$$\underline{\underline{C}} = \begin{bmatrix} \sum_{i=1}^4 \underline{\underline{C}}_i & \sum_{i=1}^4 \underline{\underline{C}}_i \underline{\underline{R}}_i \\ \sum_{i=1}^4 \underline{\underline{R}}_i^T \underline{\underline{C}}_i & \sum_{i=1}^4 \underline{\underline{R}}_i^T \underline{\underline{C}}_i \underline{\underline{R}}_i \end{bmatrix}$$

// generalized eigenvalue problem

```
[al,be,X]=spec(C,M);
```

```
alfa=zeros(6,1);
alfa2=zeros(6,1);
```

$$\underline{\underline{C}} q_{=0i} = \alpha_i^2 \underline{\underline{M}} q_{=0i} \quad (i = 1, 2, \dots, 6)$$

```
for i=1:6
```

```
    // eigenvectors normalization
```

```
    di2=X(:,i)*MM*X(:,i);
```

```
    di=sqrt(di2);
```

```
    X(:,i)=X(:,i)/di;
```

```
    // natural frequency squares
```

```
    alfa2(i,1)=X(:,i)*CC*X(:,i);
```

```
    // natural frequencies
```

```
    alfa(i,1)=sqrt(alfa2(i,1));
```

```
end
```

```
alfa_diffmethod=sqrt(al./be); //natural frequencies with a different method
```

$$q_{=ni} = \frac{q_{=0i}}{\sqrt{q_{=0i}^T \underline{\underline{M}} q_{=0i}}} \quad (i = 1, 2, \dots, 6)$$

// real part of complex numbers

```
X=real(X);
alfa2=real(alfa2);
alfa=real(alfa)
```

// generalized force

```
QF0=zeros(6,1);
QF0(1:3,1)=F;
QF0(4:6,1)=RF'*F;
```

$$\underline{\underline{Q}}_F = \begin{bmatrix} \underline{\underline{F}} \\ \underline{\underline{R}}_F^T \underline{\underline{F}} \end{bmatrix} = \begin{bmatrix} F_x \\ F_y \\ F_z \\ -z_F F_y + y_F F_z \\ z_F F_x - x_F F_z \\ -y_F F_x + x_F F_y \end{bmatrix}$$

// generalized force in the modal space

```
QF1=X'*QF0;
```

$$Q_{ai} = q_{=ni}^T \underline{\underline{Q}}$$

```
function dabi=equation(t, abi)
```

```
    fact=1;
```

```
    if t < tstart then
```

```
        fact=t/tstart;
```

```
    end
```

```
    dabi(1)=abi(2);
```

```
    dabi(2)=-2*xsi(1,1)*alfa(1,1)*abi(2)-alfa2(1,1)*abi(1)+QF1(1,1)*cos(fact*omega*t);
```

```
    dabi(3)=abi(4);
```

```
    dabi(4)=-2*xsi(2,1)*alfa(2,1)*abi(4)-alfa2(2,1)*abi(3)+QF1(2,1)*cos(fact*omega*t);
```

$$\ddot{a}_i = -2\xi_i \alpha_i \dot{a}_i - \alpha_i^2 a_i + Q_{ai} \quad (i = 1, 2, 3, 4, 5, 6)$$

```

dabi(5)=abi(6);
dabi(6)=-2*xsi(3,1)*alfa(3,1)*abi(6)-alfa2(3,1)*abi(5)+QF1(3,1)*cos(fact*omega*t);
dabi(7)=abi(8);
dabi(8)=-2*xsi(4,1)*alfa(4,1)*abi(8)-alfa2(4,1)*abi(7)+QF1(4,1)*cos(fact*omega*t);
dabi(9)=abi(10);
dabi(10)=-2*xsi(5,1)*alfa(5,1)*abi(10)-alfa2(5,1)*abi(9)+QF1(5,1)*cos(fact*omega*t);
dabi(11)=abi(12);
dabi(12)=-2*xsi(6,1)*alfa(6,1)*abi(12)-alfa2(6,1)*abi(11)+QF1(6,1)*cos(fact*omega*t);
endfunction

```

```
// differential equation solving
```

```
abi=ode(abi0,t0,t,equation);
```

```
// sorting displacement and velocity
```

```
for i=1:6
```

```
    ai(i,:)=abi(2*i-1,:);    // normalized displacement
```

```
    bi(i,:)=abi(2*i,:);    // normalized velocity
```

```
end
```

```
// general displacement coordinates
```

```
qi=X*ai;
```

```
uvw=qi(1:3,:);
```

```
fixyz=qi(4:6,:);
```

```
// displacement of point P
```

```
uvwP=uvw+RP*fixyz;
```

$$\underline{u}_P = \underline{u}_S + \underline{R}_P \underline{\varphi}$$

```
subplot(2,1,1);
```

```
plot(t,uvwP(3,:)*1000)
```

```
xtitle("Displacement function of point P", "time (s)", "displacement (mm)")
```

```
xgrid(2)
```

```
qdi=X*bi;
```

```
duvw=qdi(1:3,:);
```

```
dfixyz=qdi(4:6,:);
```

```
// velocity of point P
```

```
duvwP=duvw+RP*dfixyz;
```

$$\underline{\dot{u}}_P = \underline{\dot{u}}_S + \underline{R}_P \underline{\dot{\varphi}}$$

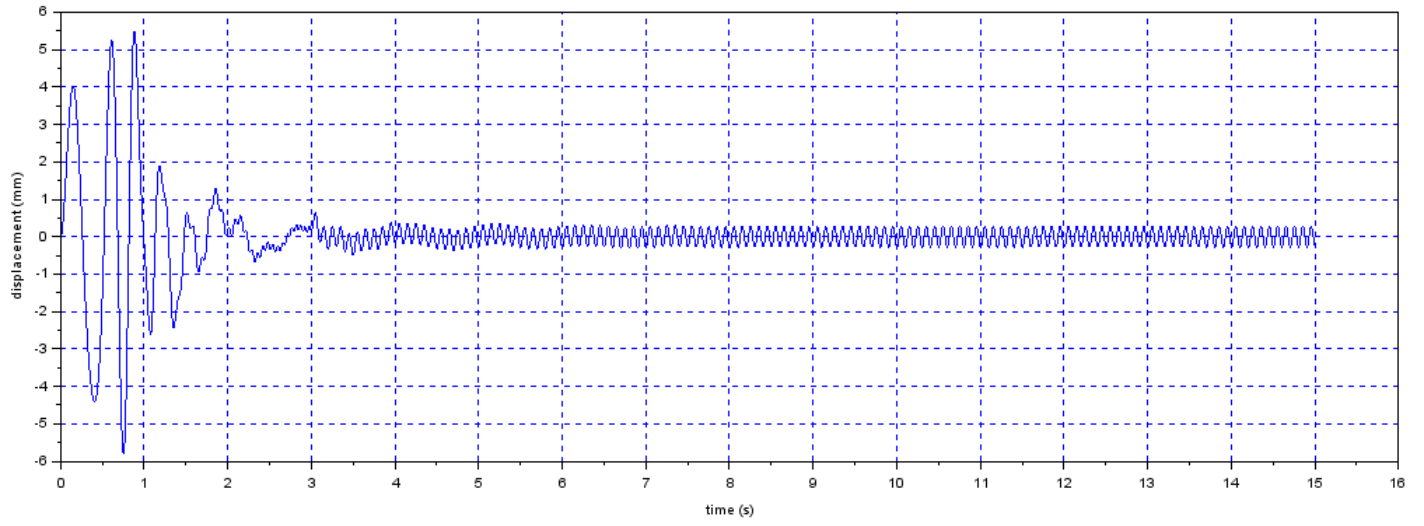
```
subplot(2,1,2);
```

```
plot(t,duvwP(3,:))
```

```
xtitle("Velocity function of point P ", "time (s)", "velocity (m/s)")
```

```
xgrid(2)
```

Displacement function of point P



Velocity function of point P

