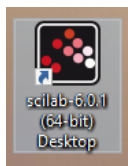
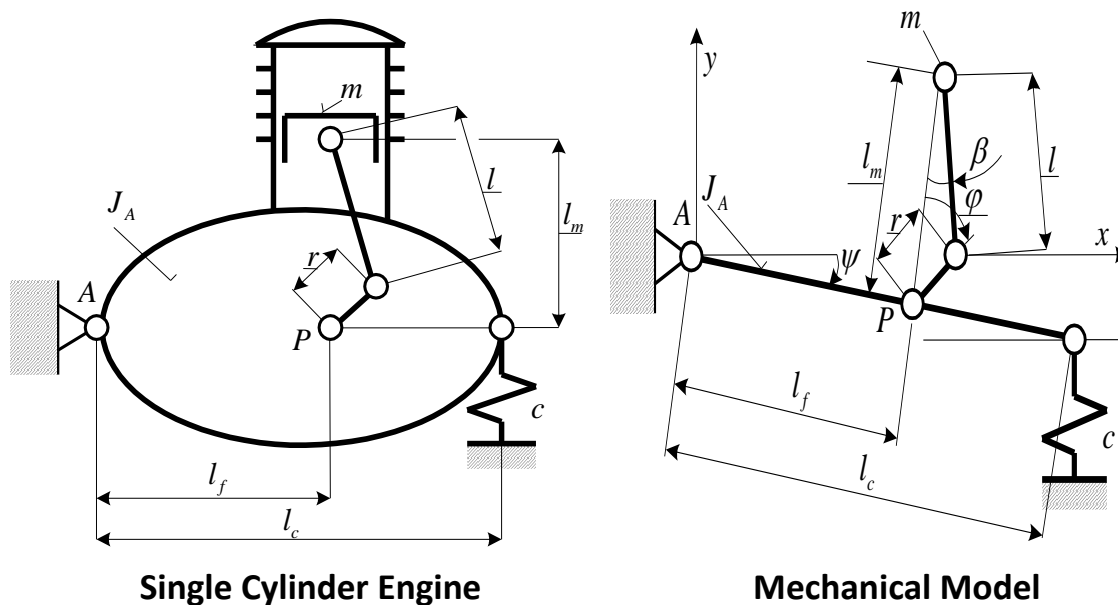


Dynamics of Machines Week 7 – Exercise



Vibrations of a Single Cylinder Engine with Elastic Mounting



Single Cylinder Engine

Mechanical Model

Equation of motion:

$$\underbrace{\left[J_{Az} + m \left(l_f^2 + l^2 + \frac{r^2}{2} \right) \right]}_{J_{red}} \ddot{\psi} + \frac{l_c^2}{c} \dot{\psi} = -ml_f r \Omega^2 \cos \Omega t, \quad \psi = \frac{y_p}{l_f}, \quad \ddot{\psi} = \frac{\ddot{y}_p}{l_f}$$

Rearrangement:

$$\ddot{y}_p = -\frac{l_c^2}{c J_{red}} y_p - \frac{ml_f^2 r}{J_{red}} \Omega^2 \cos \Omega t$$

$$\alpha^2$$

If we take into account damping: $\ddot{y}_p = -2\alpha\xi\dot{y}_p - \alpha^2 y_p - \frac{ml_f^2 r}{J_{red}} \Omega^2 \cos \Omega t$

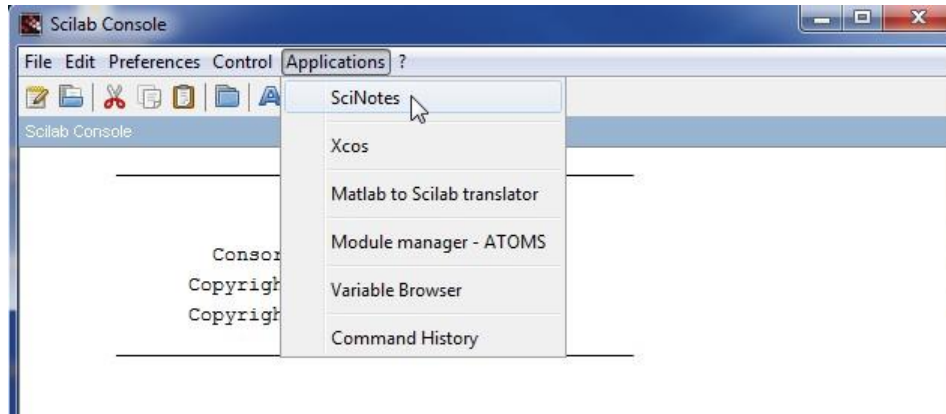
(where $\xi = 0, 1$ is the material damping)

Idle speed of the engine is $n_{\min} = 1000 \text{ ford} / \text{min}$, maximum engine speed is $n_{\max} = 5000 \text{ ford} / \text{min}$. Determine the spring constant $c \text{ (m} / \text{N)}$ of the engine mount that the vertical displacement of point P must stay in the $\pm 1 \text{ mm}$ range in both maximum and minimum engine speed!

There are two spring constants that are the results of previous calculation.

$c_1 = 0,00281 \text{ N/m}$ (soft spring) and $c_2 = 0,000102 \text{ N/m}$ (stiffer spring)

Determine which spring is suitable for the engine to fulfill the requirement ($\pm 1 \text{ mm}$)!



// Vibration of a Single Cylinder Engine (omega is variable, Euler method)

```
clear;
// Variables
lc=0.35; lf=0.19;           // (m)
r=0.03;                    // crank radius (m)
l=0.12;                    // connecting rod length (m)
Jaz=0.52;                  // moment of inertia of all parts about z axis (kgm^2)
m_pist=0.5;               // piston mass (kg)
Jred_engine=(Jaz+m_pist*(lf^2+l^2+r^2/2)); // reduced moment of inertia about z axis (kgm^2)
Jred_pist=m_pist*(lf^2+l^2+r^2/2); // reduced moment of inertia of piston about z axis (kgm^2)
// c=0.00281; // (m/N) spring constant, of the soft spring
c=0.000112; // (m/N) spring constant of the stiff spring
kszi=0.1; // Lehr damping coefficient
alf2=lc^2/(c*Jred_engine);
alf=sqrt(alf2); // (rad/s) undamped natural frequency
M0=8; // (Nm) engine torque
// Engine speed values
n_min=1000; // min engine speed (rev/min)
om_min=2*pi*n_min/60; // min angular velocity (rad/s)
n_max=5000; // max engine speed (ford/min)
om_max=2*pi*n_max/60; //max angular velocity (rad/s)
////////////////////////////////////
tmax=10; // end time (s)
dt=0.0005; // time step (s)
n=int(tmax/dt); // number of time steps
// Initial values
t0=-dt; // (s)
y0=0; // (m)
v0=0; // (m/s)
a0=0; // (m/s^2)
```

```

omega0=0;           // (rad/s)
epsz0=M0/Jred_pist; // (rad/s^2)
a=(1:n);
v=(1:n);
y=(1:n);
om=(1:n);
epsz=(1:n);
// Calculation ////////////////////////////////////////////////////
// Min. engine speed – black color //
for i=1:n
    t(i)=t0+dt;
    epsz(i)=M0/Jred_pist;
    om(i)=omega0+((epsz(i)+epsz0)/2)*dt;

if om(i)<om_min then
    om1=om(i);
    omeg(i)=om(i);           // It will be plotted
    else
    om1=om_min;
    omeg(i)=om_min;         // It will be plotted
end
a(i)=(-2*alf*kszi*v0-alf2*y0-(m_pist*lf^2*r*om1^2/Jred_engine)*cos(om1*t(i)));
v(i)=v0+((a(i)+a0)/2)*dt;
y(i)=y0+((v(i)+v0)/2)*dt;
// Variables value exchange
t0=t(i);
v0=v(i);
y0=y(i);
a0=a(i);
omega0=om1;
epsz0=epsz(i);
end

// Plotting ////////////////////////////////////////////////////
subplot(2,1,1)
plot2d(t,60*omeg/(2*pi),1)
xlabel("Engine speed", " t (s)", "n (rev/min)" )
subplot(2,1,2)
plot2d(t,y*1000,1)
xlabel("Vertical displacement of point P ", " t (s)", "yp (mm)" )
xgrid(2);

// Copy the script from the initial conditions to the end and paste it below and change some parameters and
variables.

// Max engine speed – red color ////////////////////////////////////////////////////
t0=-dt;
y0=0;
v0=0;
a0=0;
omega0=0;
epsz0=M0/Jred_pist;
for i=1:n
    t(i)=t0+dt;

```

```

epsz(i)=M0/Jred_pist;
om(i)=omega0+((epsz(i)+epsz0)/2)*dt;

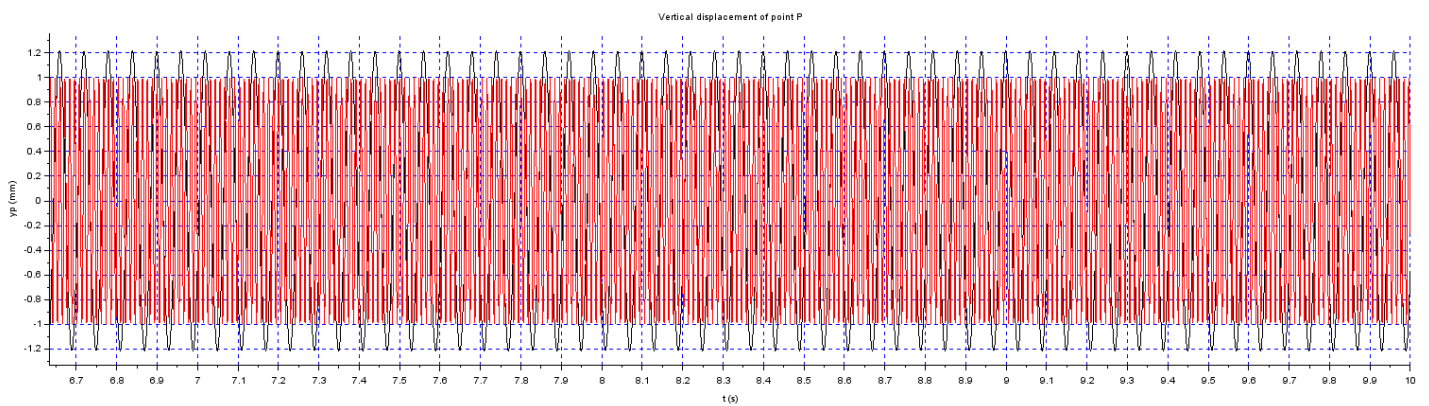
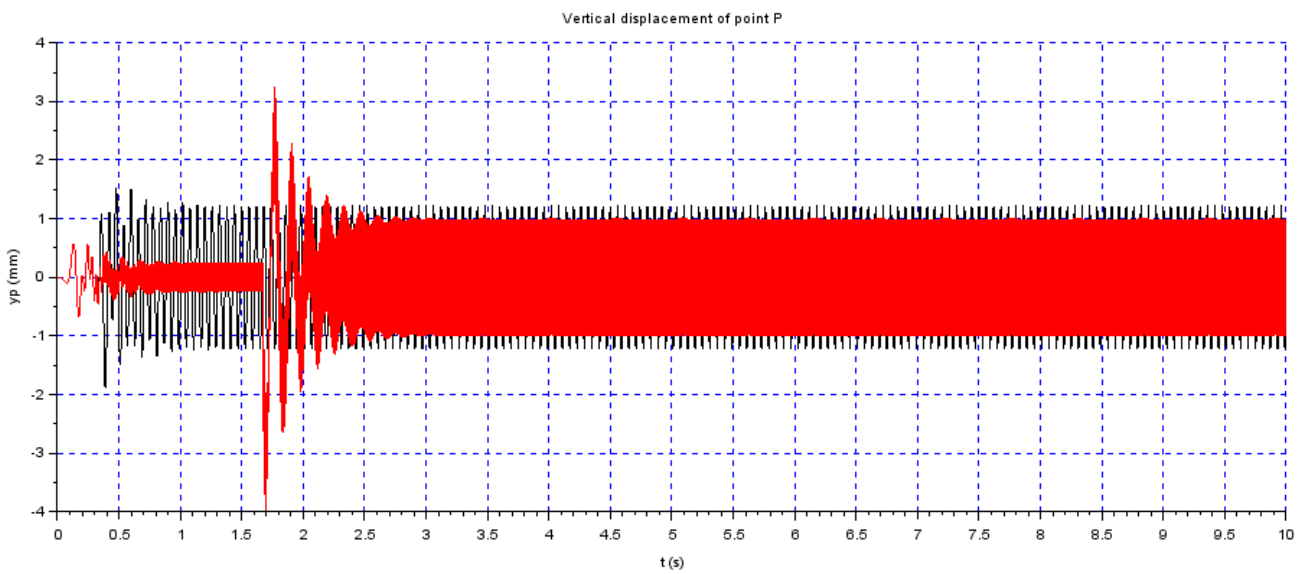
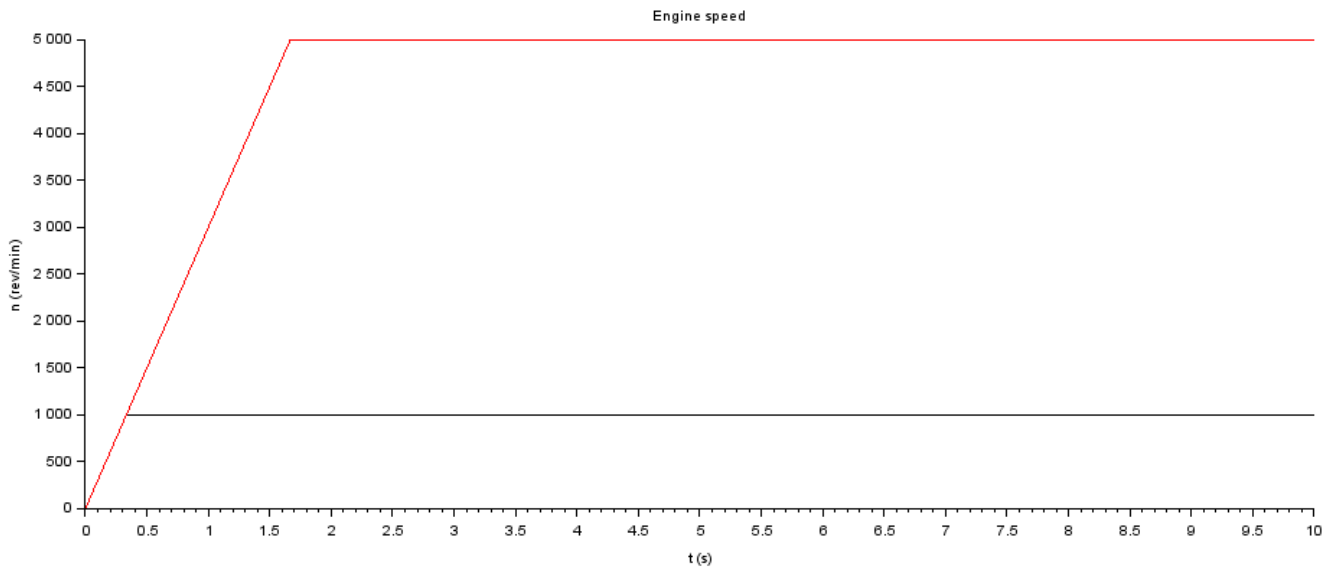
if om(i)<om_max then
    om1=om(i);
    omeg(i)=om(i);           // It will be plotted
    else
    om1=om_max;
    omeg(i)=om_max;         // It will be plotted
    end
a(i)=(-2*alf*kszi*v0-alf2*y0-(m_pist*lf^2*r*om1^2/Jred_engine)*cos(om1*t(i)));
v(i)=v0+((a(i)+a0)/2)*dt;
y(i)=y0+((v(i)+v0)/2)*dt;

t0=t(i);
v0=v(i);
y0=y(i);
a0=a(i);
omega0=om1;
epsz0=epsz(i);
end
// Plotting
subplot(2,1,1)
plot2d(t,60*omeg/(2*pi),5)
xlabel("Engine speed", " t (s)", "n (rev/min)" )
subplot(2,1,2)
plot2d(t,y*1000,5)
xlabel("Vertical displacement of point P ", " t (s)", "yp (mm)" )
xgrid(2);

```

>>>>> Save the script.

Execute: Execute >>>> file with echo

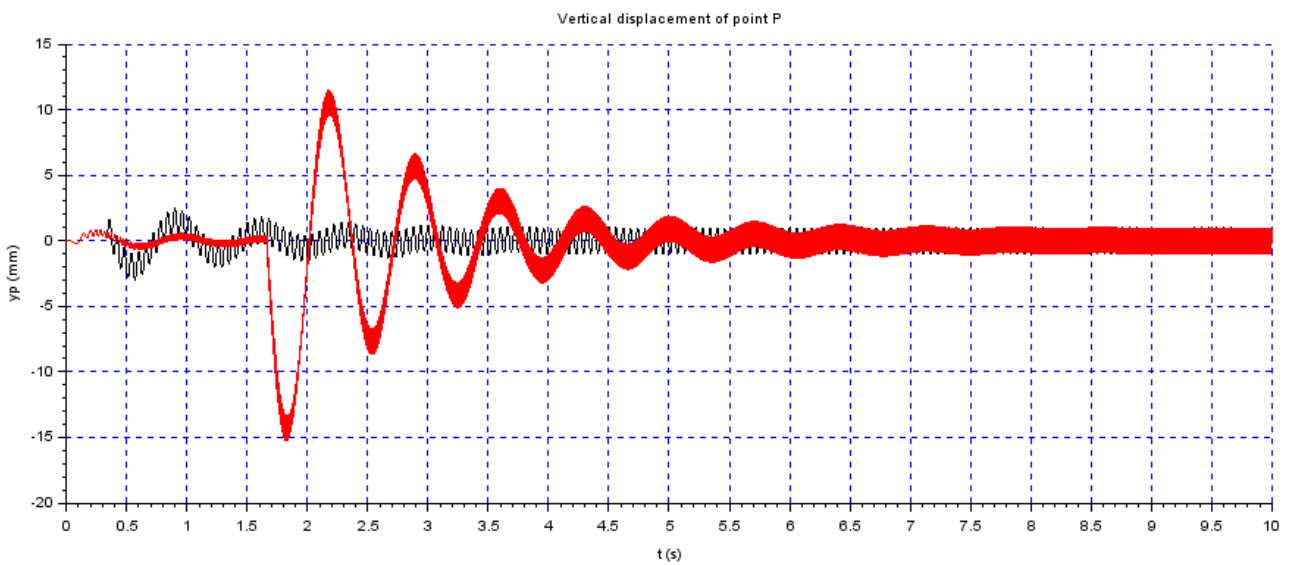
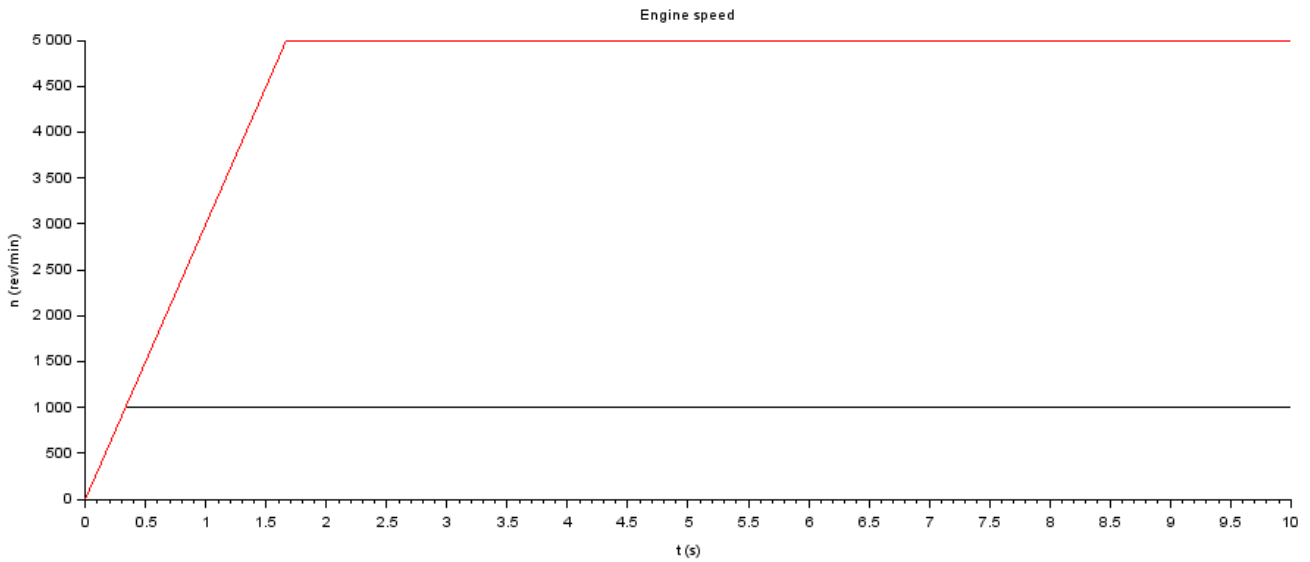


Analyze the steady state vibration after $t = 8$ s

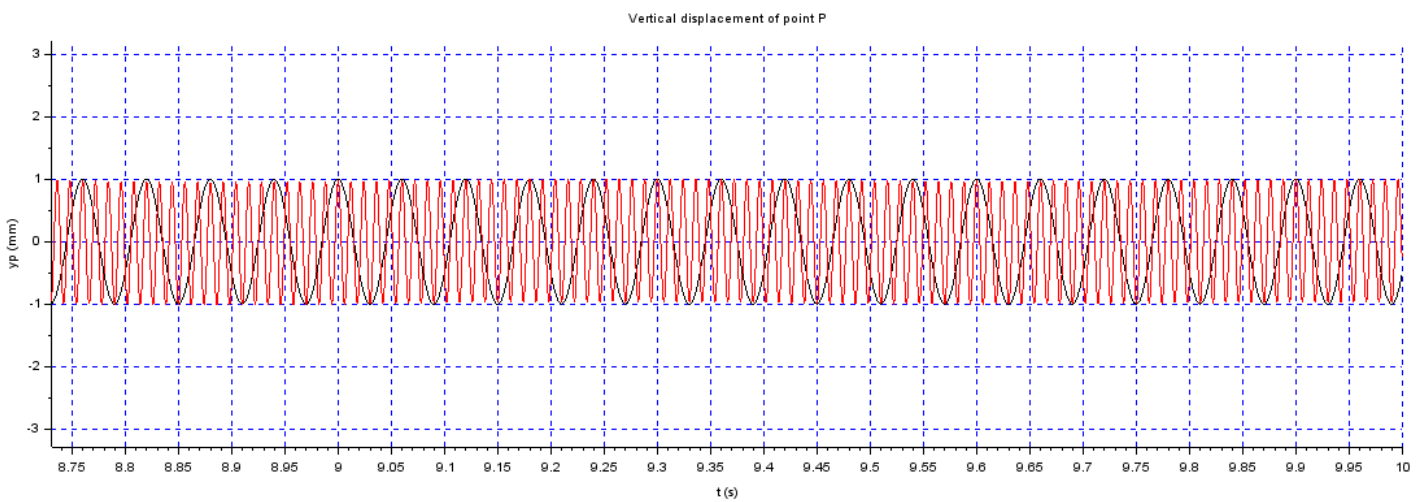
The stiffer spring is not suitable for the engine because in idle speed the vertical displacement is higher than $(+/-) 1$ mm

Close the graphic window and change the spring constant to $c = 0.00281$ m/N

Execute (with soft spring): Execute >>>> file with echo



Analyze the steady state vibration after $t = 8$ s



The soft spring is the suitable one because in idle speed and also in maximum engine speed the vertical displacement of point P is in the (+/-) 1mm range.